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**Bacon**

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(54) **BELT-DRIVEN TRANSPORTATION SYSTEM**

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4,872,800 A	10/1989	Gutov et al.
4,899,987 A	2/1990	Craig
4,905,496 A	3/1990	Venalainen
4,921,074 A	5/1990	Ochs
5,149,242 A	9/1992	Haymore
5,299,906 A	4/1994	Stone
5,322,143 A	6/1994	Curran
5,513,723 A	5/1996	Luebke
5,636,711 A	6/1997	Nussbaum
5,695,173 A	12/1997	Ochoa et al.
5,722,513 A	3/1998	Rowan et al.

(Continued)

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187/269

See application file for complete search history.

(56) **References Cited**

**U.S. PATENT DOCUMENTS**

484,038 A *	10/1892	Morgan	254/399
1,991,255 A	2/1935	Martin	
3,785,462 A *	1/1974	Coad et al.	187/262
3,901,356 A	8/1975	Butler	
4,219,186 A	8/1980	Brewer	
4,427,093 A	1/1984	Wehmeyer et al.	
4,534,544 A	8/1985	Heide	
4,577,821 A	3/1986	Edmo et al.	
4,619,346 A	10/1986	Deguerry	
4,858,482 A	8/1989	Knudsen	

**FOREIGN PATENT DOCUMENTS**

EP 0501254 A2 2/1992

**OTHER PUBLICATIONS**

English language translation of Abstract for European Patent Application No. 0501254A2, filed Feb. 28, 1991.

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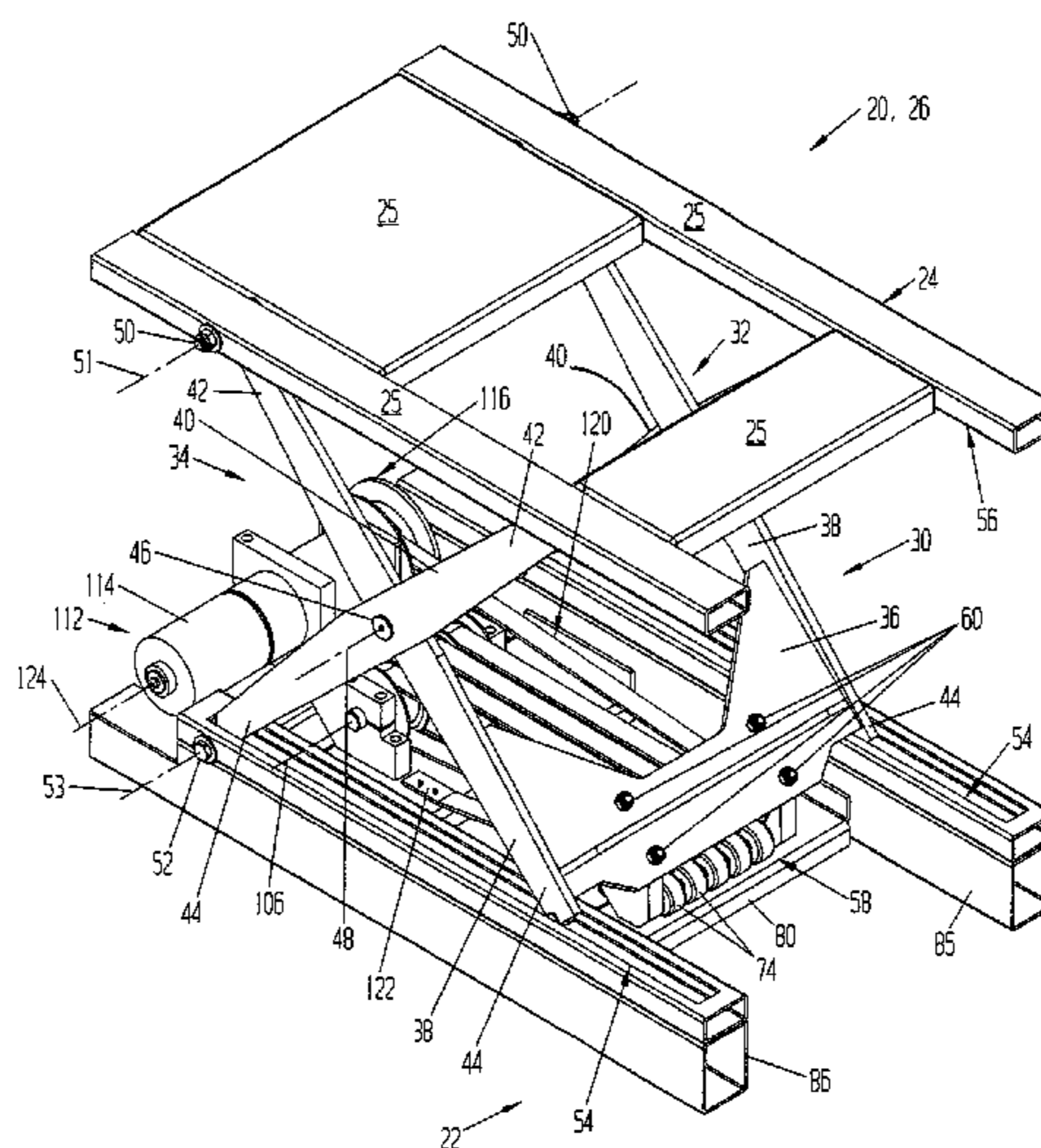
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(57) **ABSTRACT**

A belt-driven transportation system including a first set of pulleys rotatably attached to a first member and a second set of pulleys rotatably attached to a second member. The first and second members have relative movement to each other. The system further includes a unitary belt that is guided through a path defined by the first and second sets of pulleys with the path substantially lying in a stationary plane, and a plurality of retention rollers. A retention roller is disposed adjacent each pulley, with the belt extending about the pulley and between the pulley and the retention roller. The positioning of the belt substantially on the path is maintained by the plurality of retention rollers, and the first and second members are moved relative to each other with changes in the length of the path.

**17 Claims, 13 Drawing Sheets**



(56)

**References Cited**

U.S. PATENT DOCUMENTS

5,934,414	A	8/1999	Staczek	6,705,238	B1	3/2004	Heckert	
6,092,788	A	7/2000	Simon	6,742,768	B2	6/2004	Alba	
6,257,372	B1	7/2001	Schirmer	6,779,635	B1	8/2004	Anibas	
6,286,629	B1	9/2001	Saunders	6,814,188	B1	11/2004	Heckert	
6,305,499	B1	10/2001	Jones et al.	6,854,715	B2	2/2005	Hicks et al.	
6,371,247	B1	4/2002	Nedderman, Jr.	7,070,167	B1	7/2006	Bacon et al.	
6,464,205	B2	10/2002	Wanner	7,117,977	B2	10/2006	Abe	
6,537,017	B2	3/2003	Stone	7,213,686	B2	5/2007	Kaufman	
6,591,945	B1	7/2003	Kigawa et al.	7,225,901	B2	6/2007	Mustalahti et al.	
6,601,826	B1	8/2003	Granata	7,331,425	B2	2/2008	Bukowski et al.	
6,619,433	B1	9/2003	Robert et al.	7,383,923	B2	6/2008	Patten et al.	
6,634,462	B2	10/2003	Byeong-Ho	7,413,056	B2 *	8/2008	Gonzi et al. ....	187/269
6,640,934	B1	11/2003	Edwards	7,461,722	B2	12/2008	Kyotani	
6,672,430	B2	1/2004	Boucher et al.	7,506,725	B2	3/2009	Kuo et al.	
6,679,479	B1	1/2004	Watkins	2006/0037518	A1	2/2006	Lopez Alba	
				2007/0213154	A1 *	9/2007	Broyan .....	474/148
				2011/0139549	A1 *	6/2011	Bacon .....	187/269

\* cited by examiner

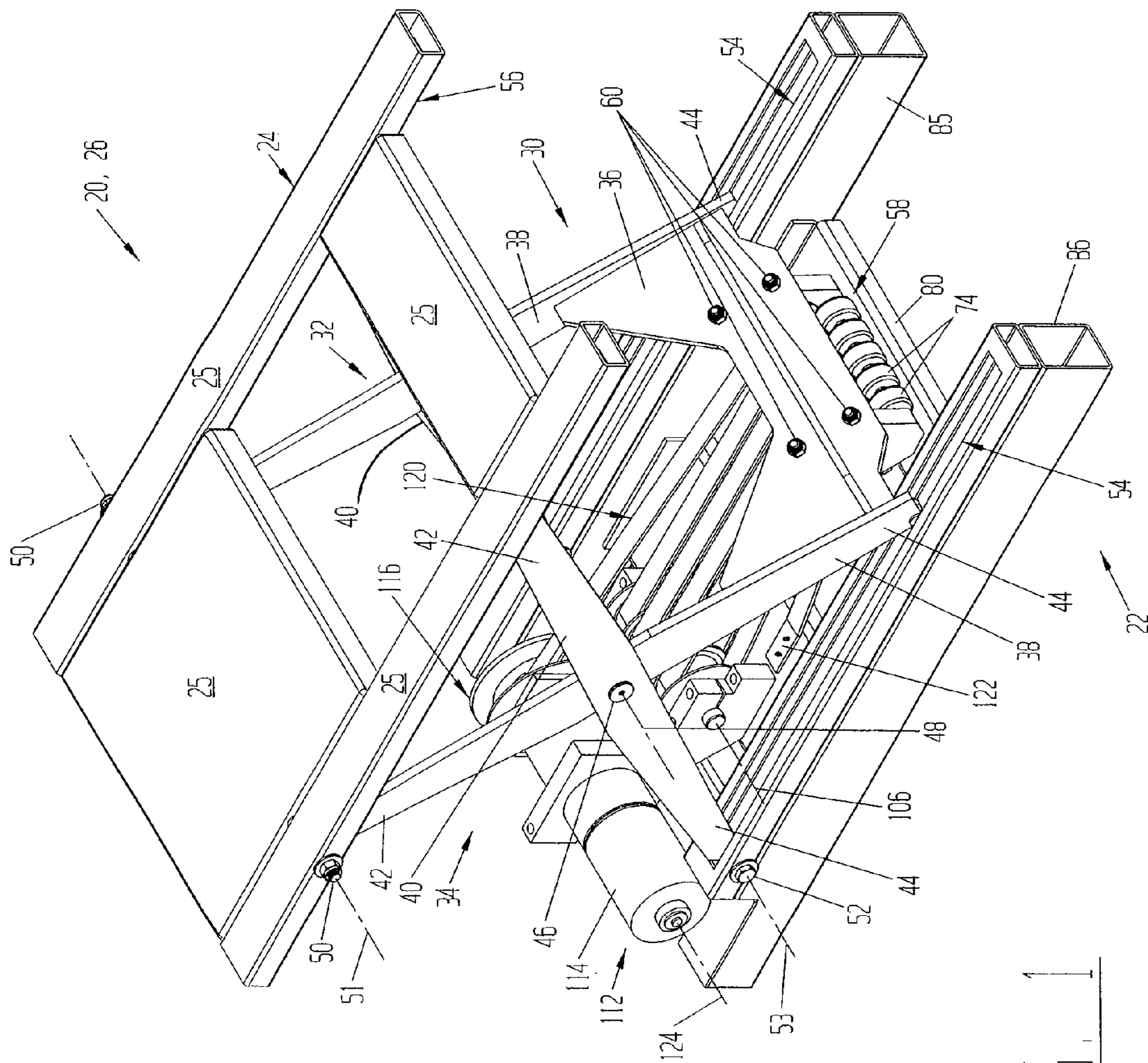


FIG. 1

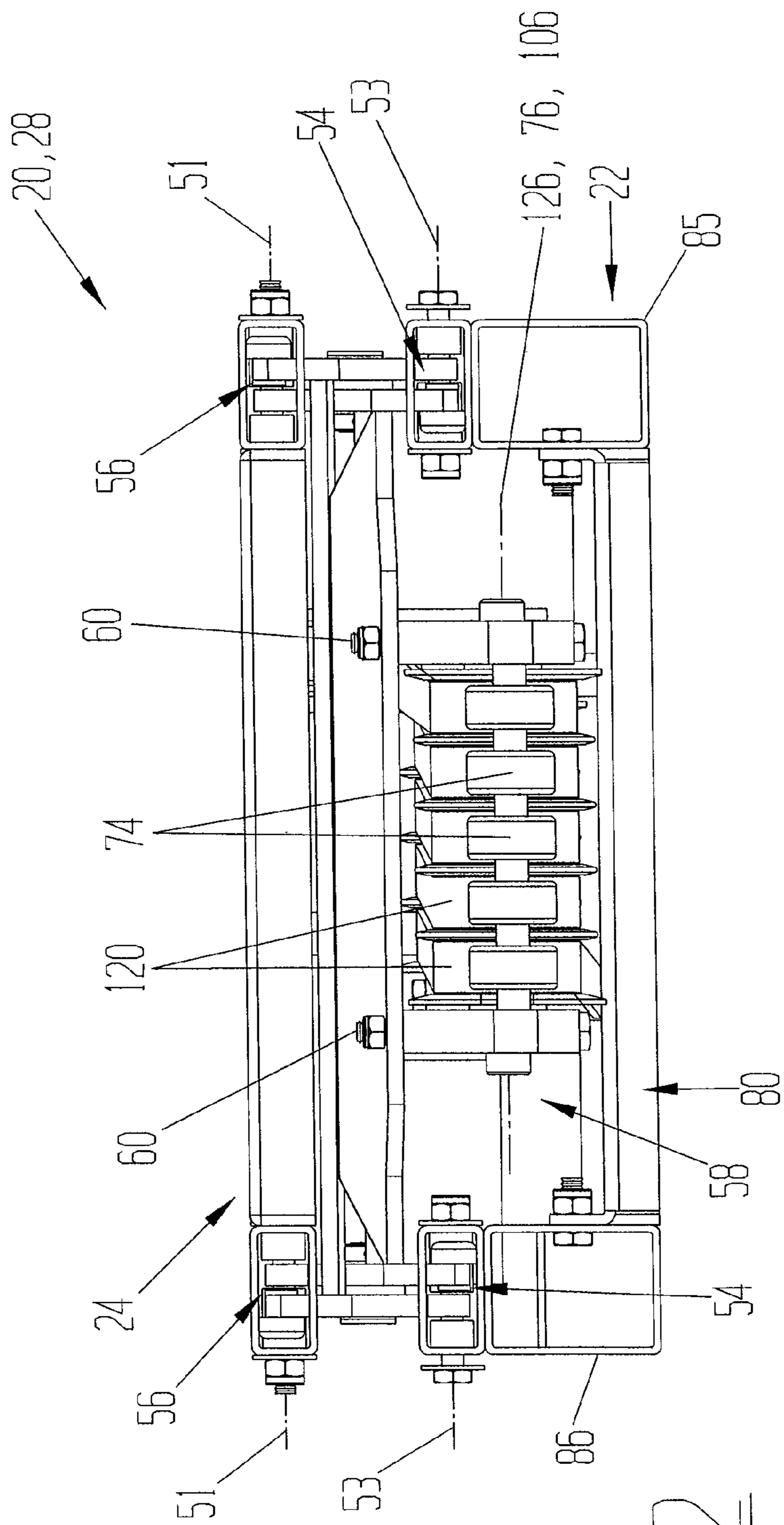


FIG. 2

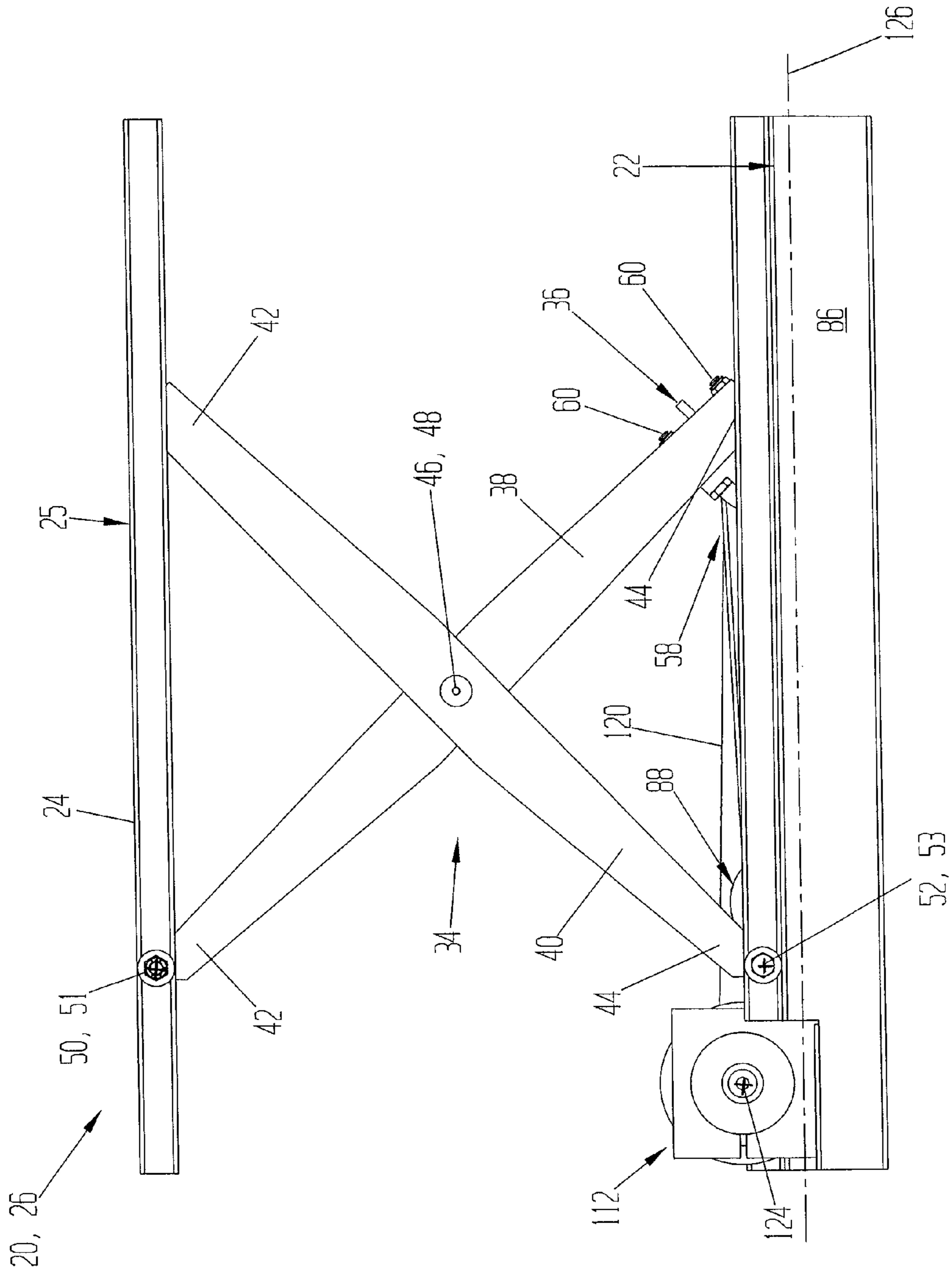


FIG. 3

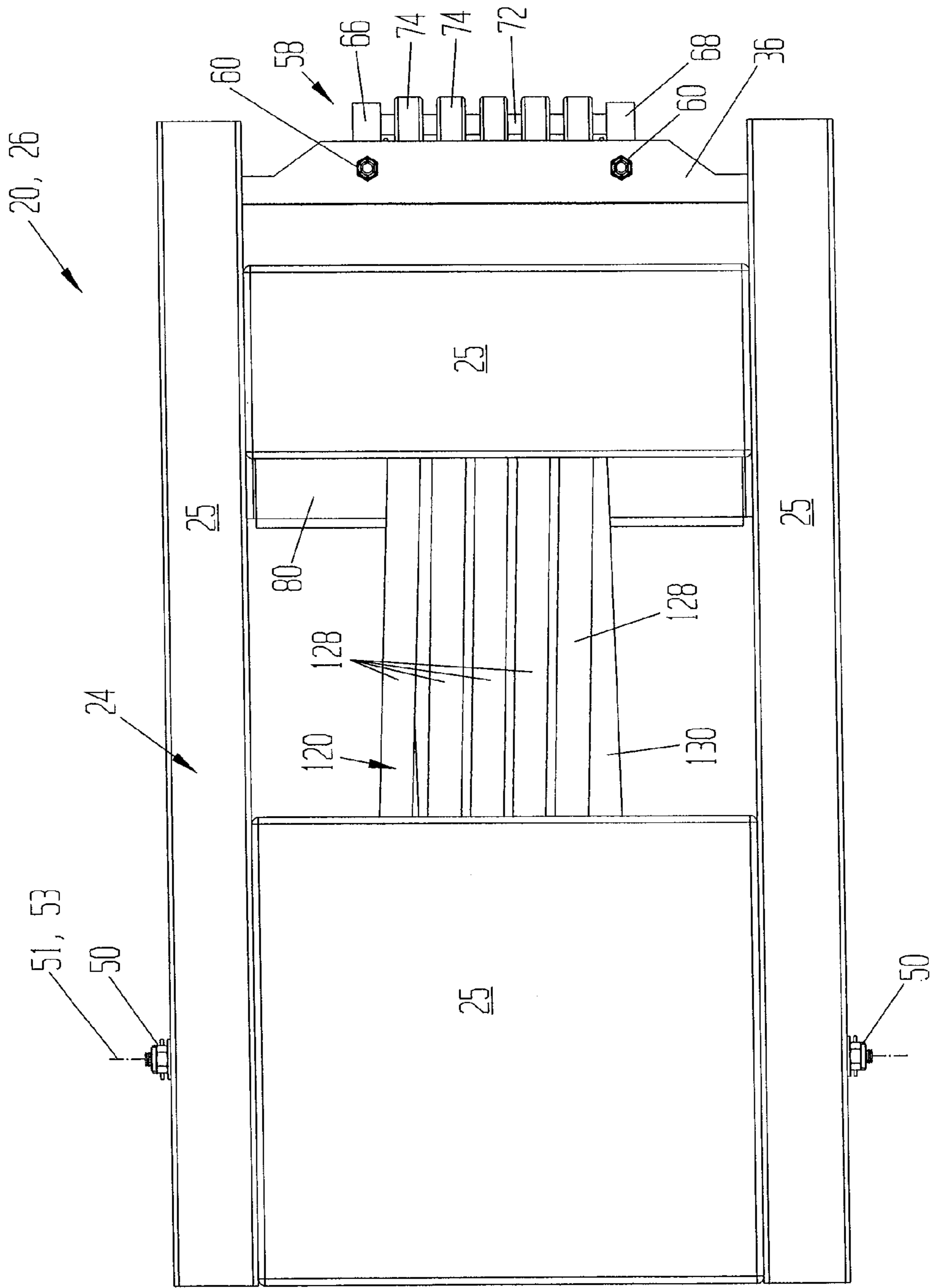


FIG. 4

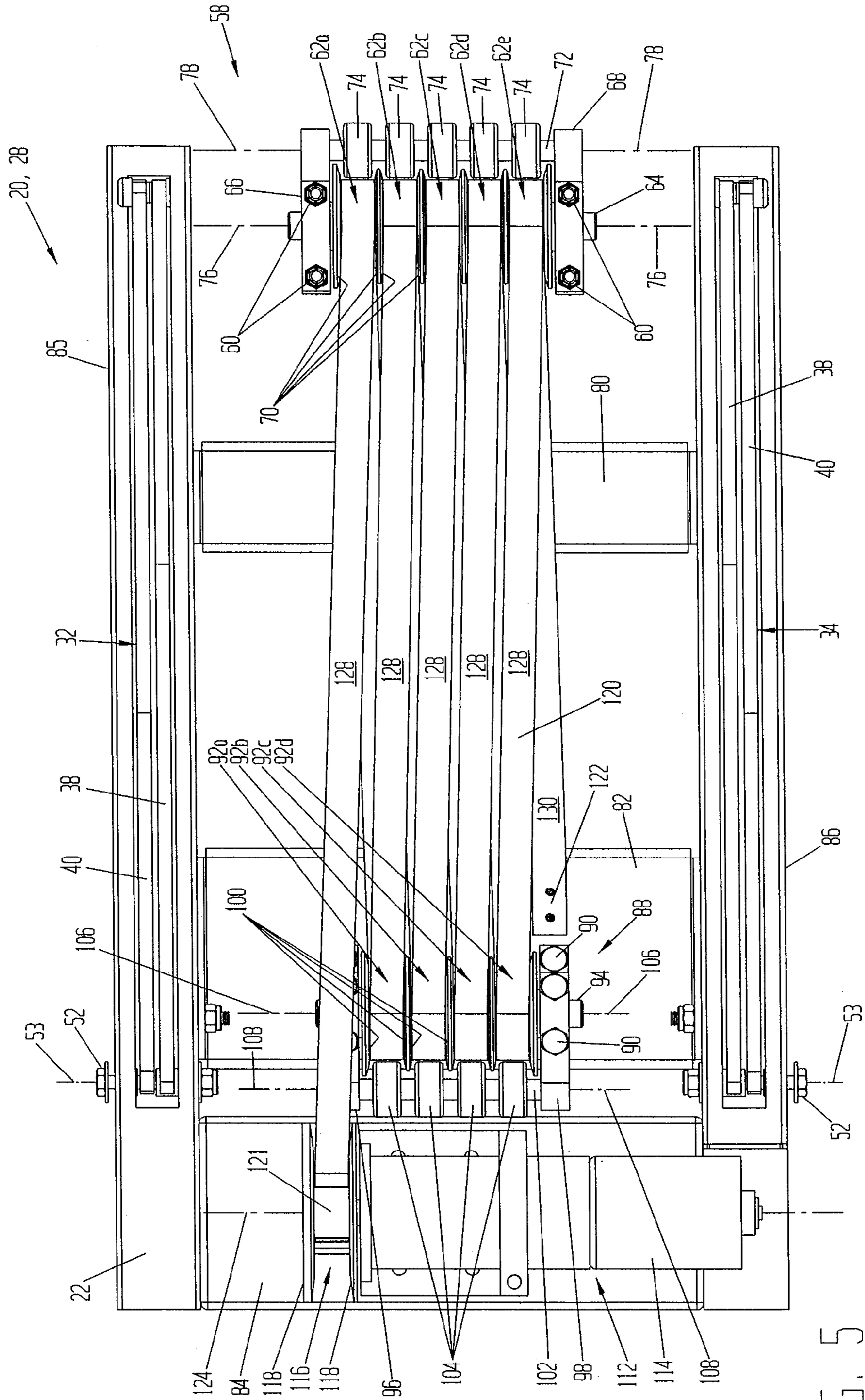


FIG. 5

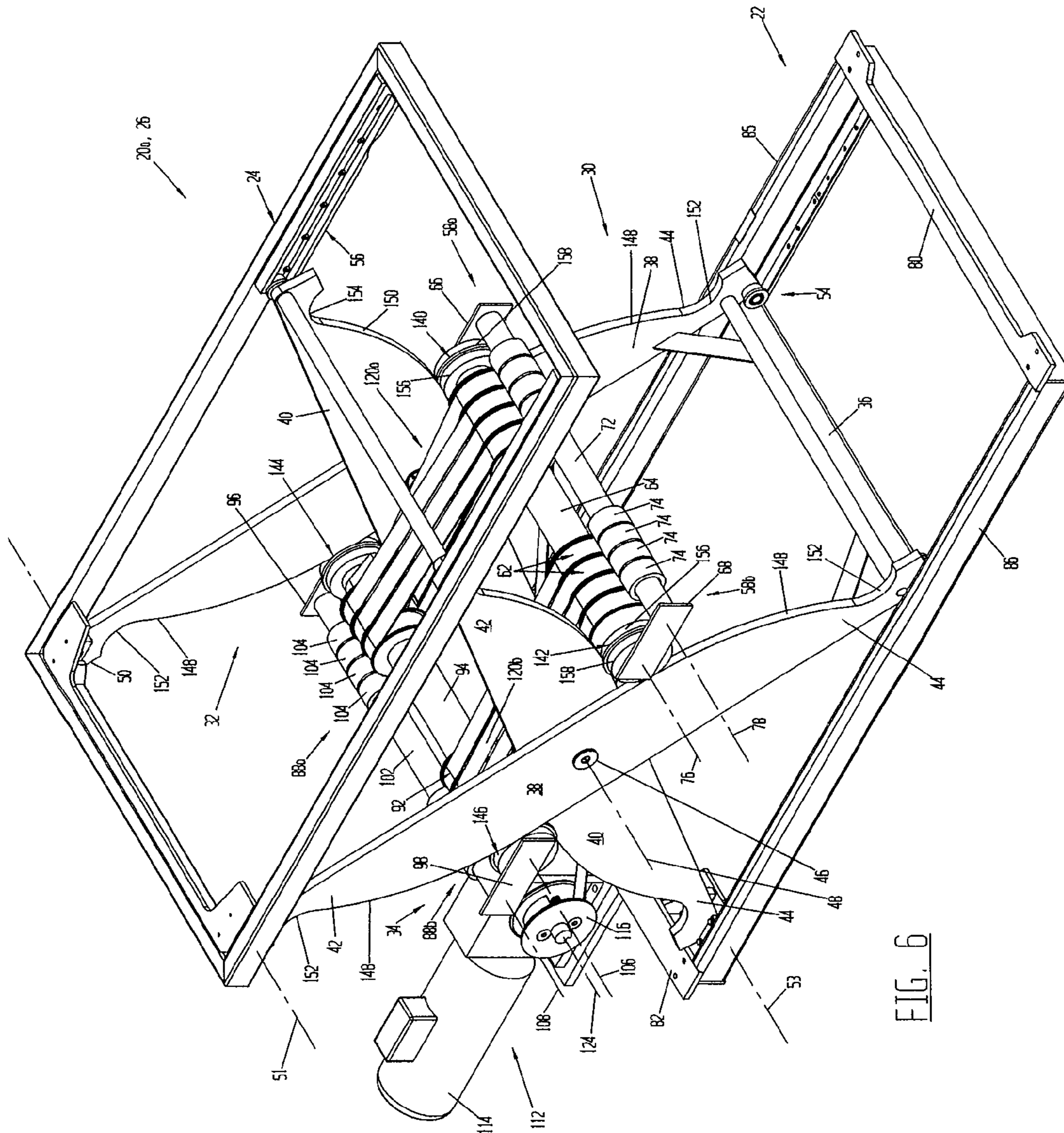


FIG. 6



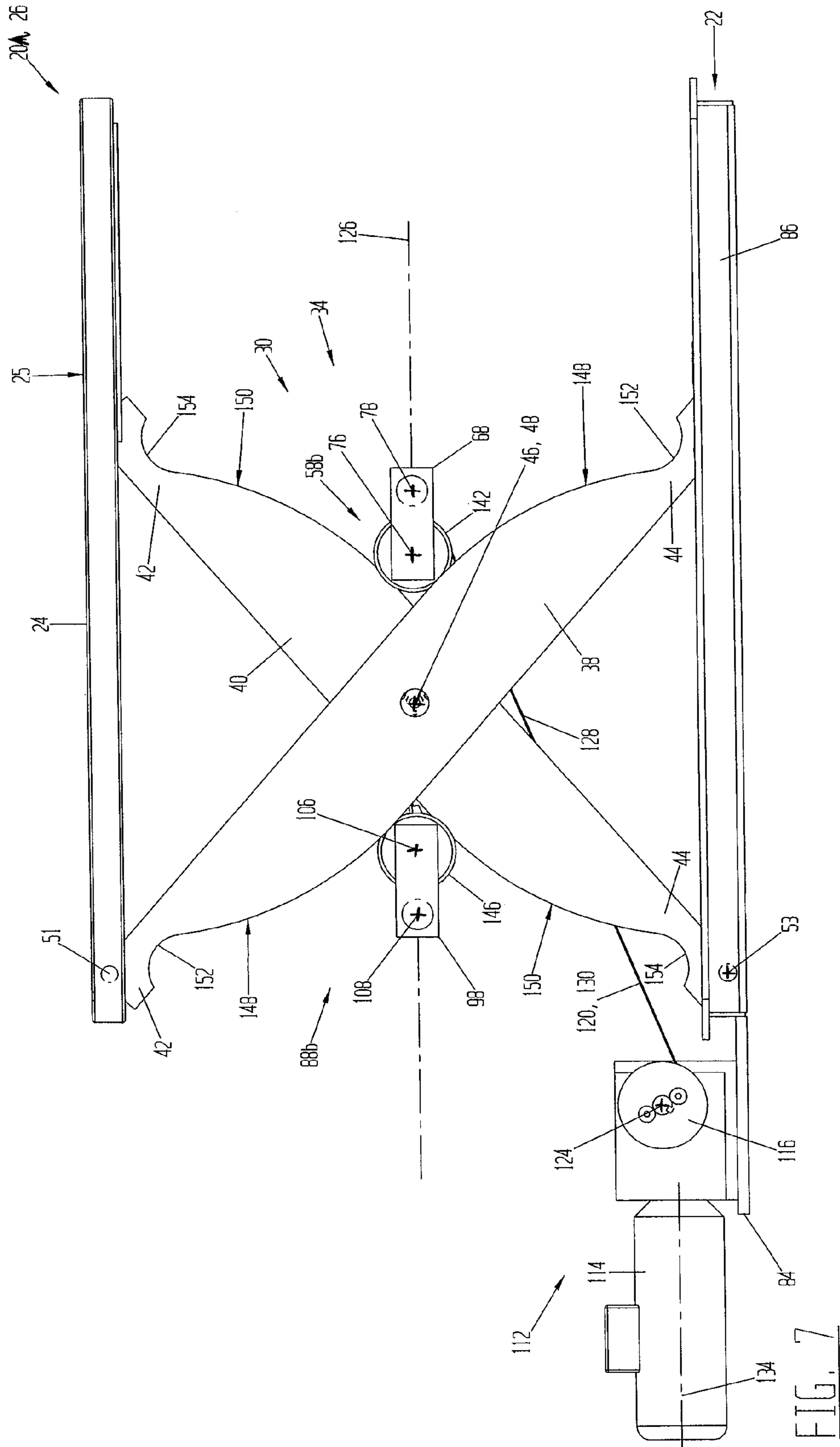


FIG. 7

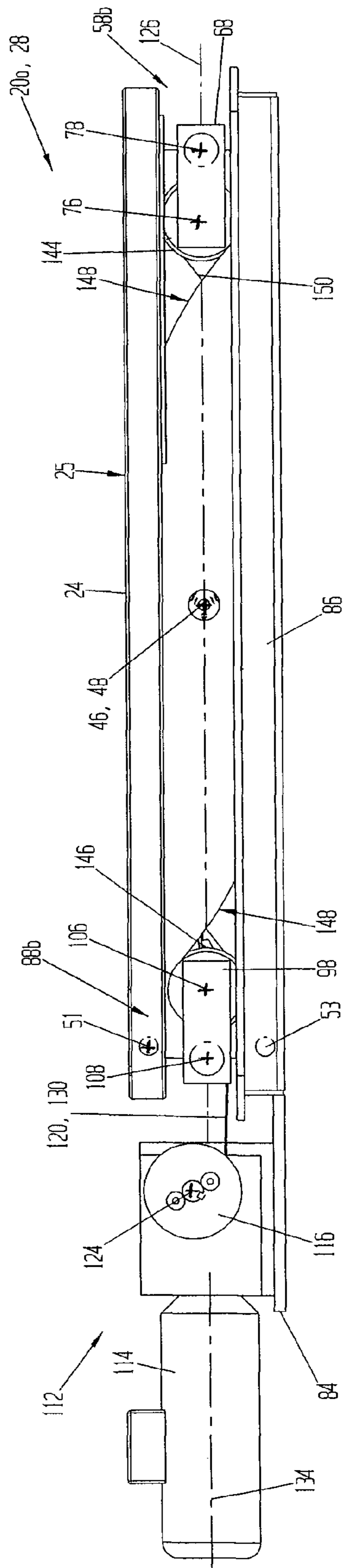
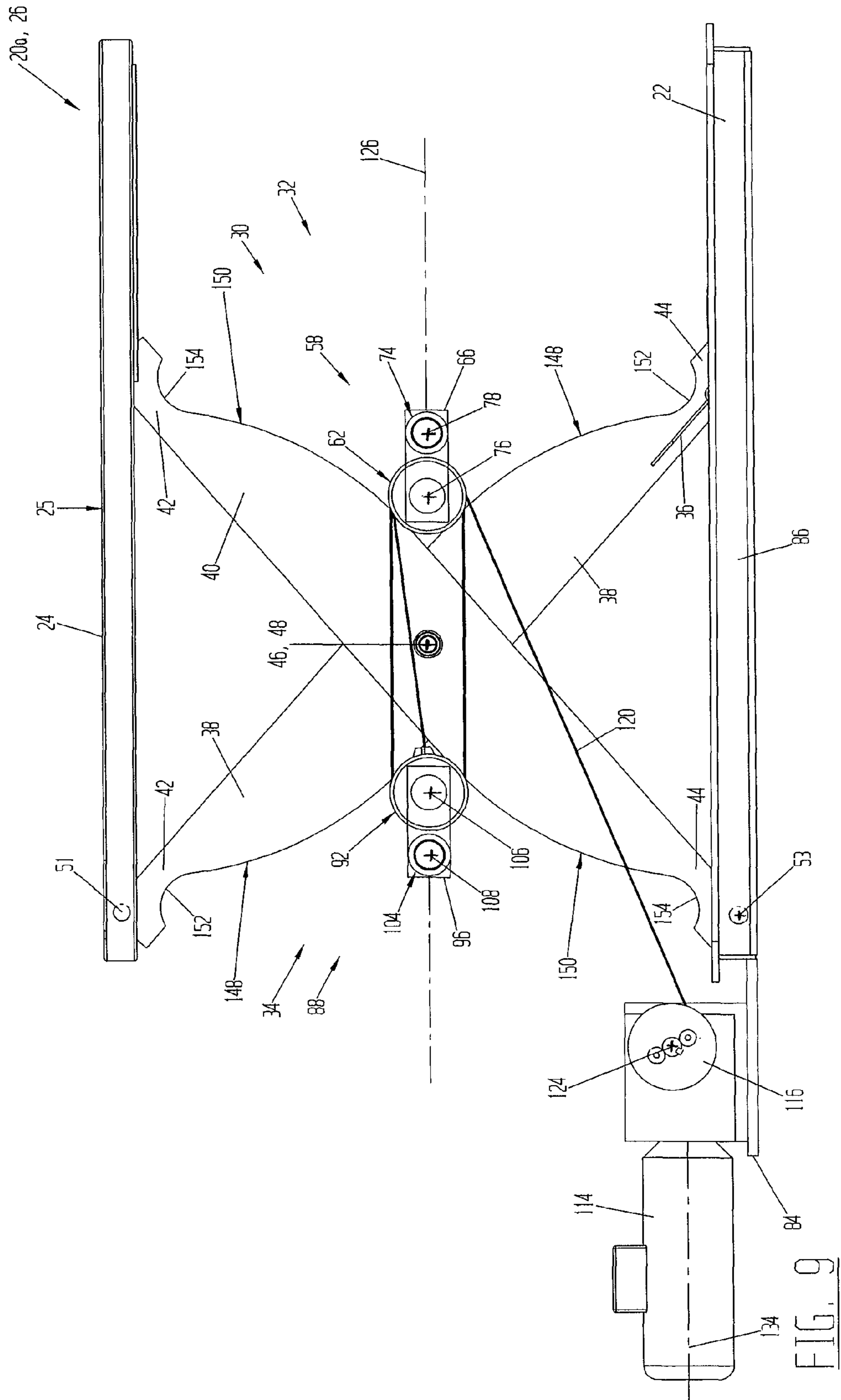


FIG. 8



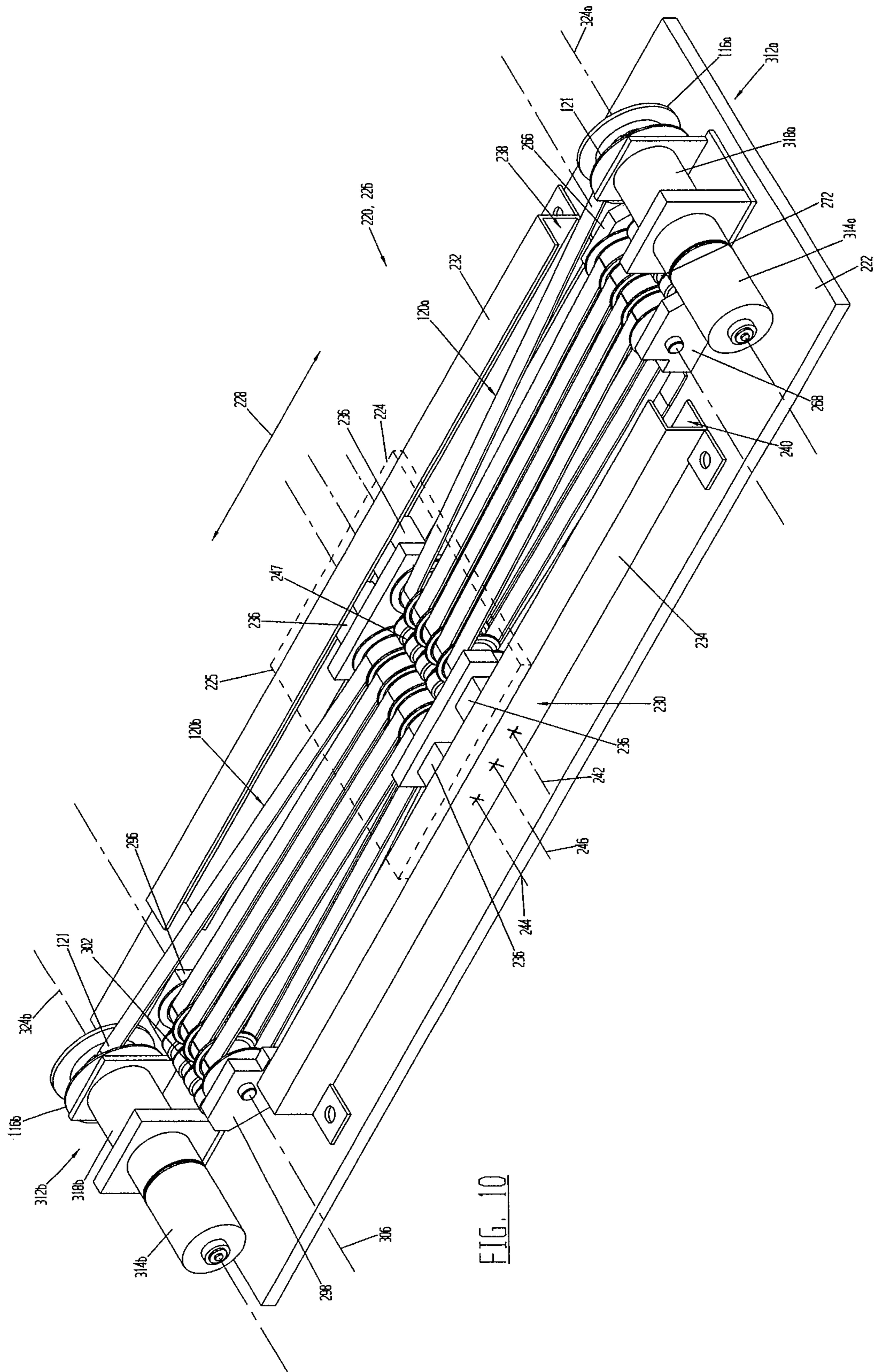


FIG. 10

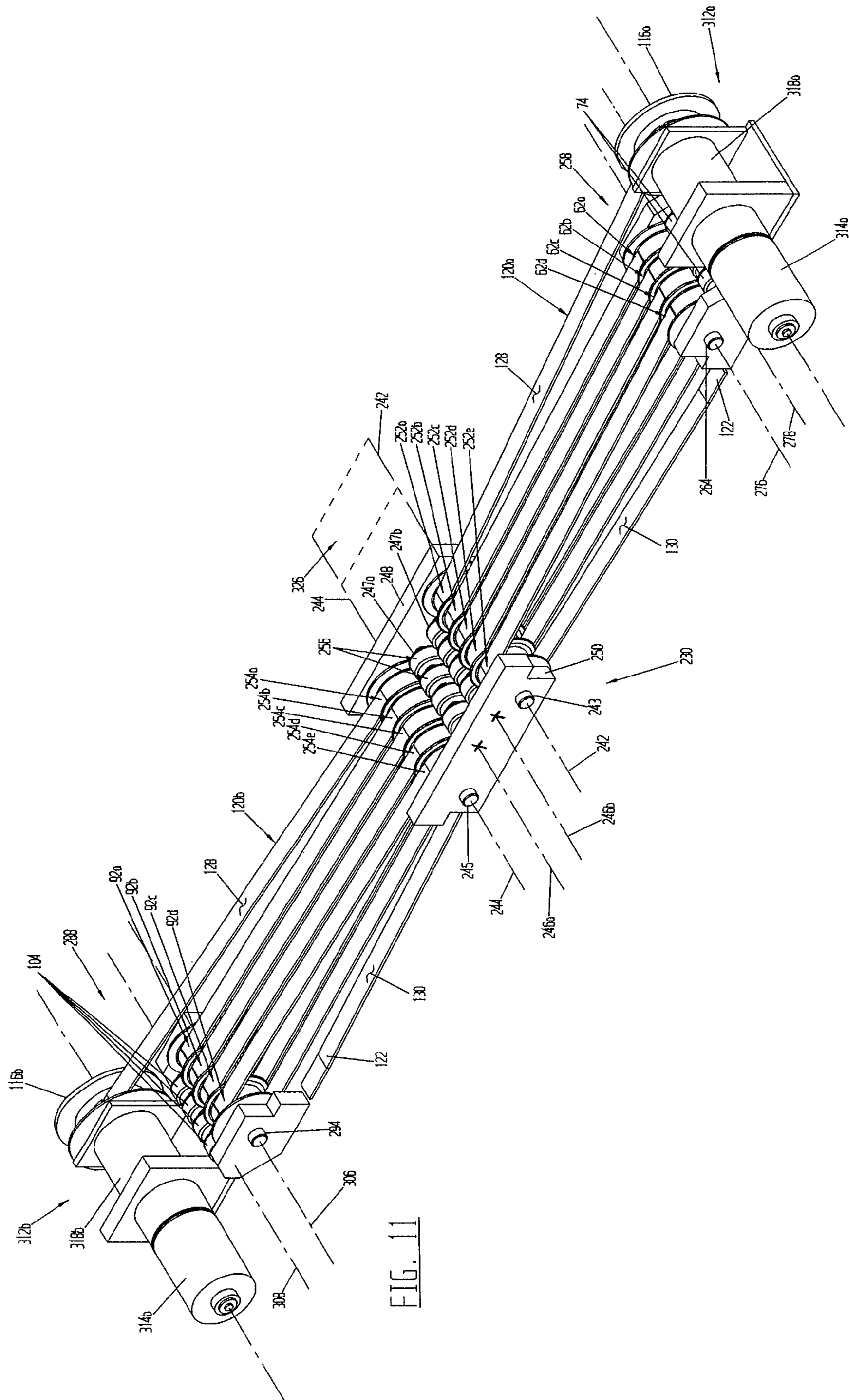


FIG. 11

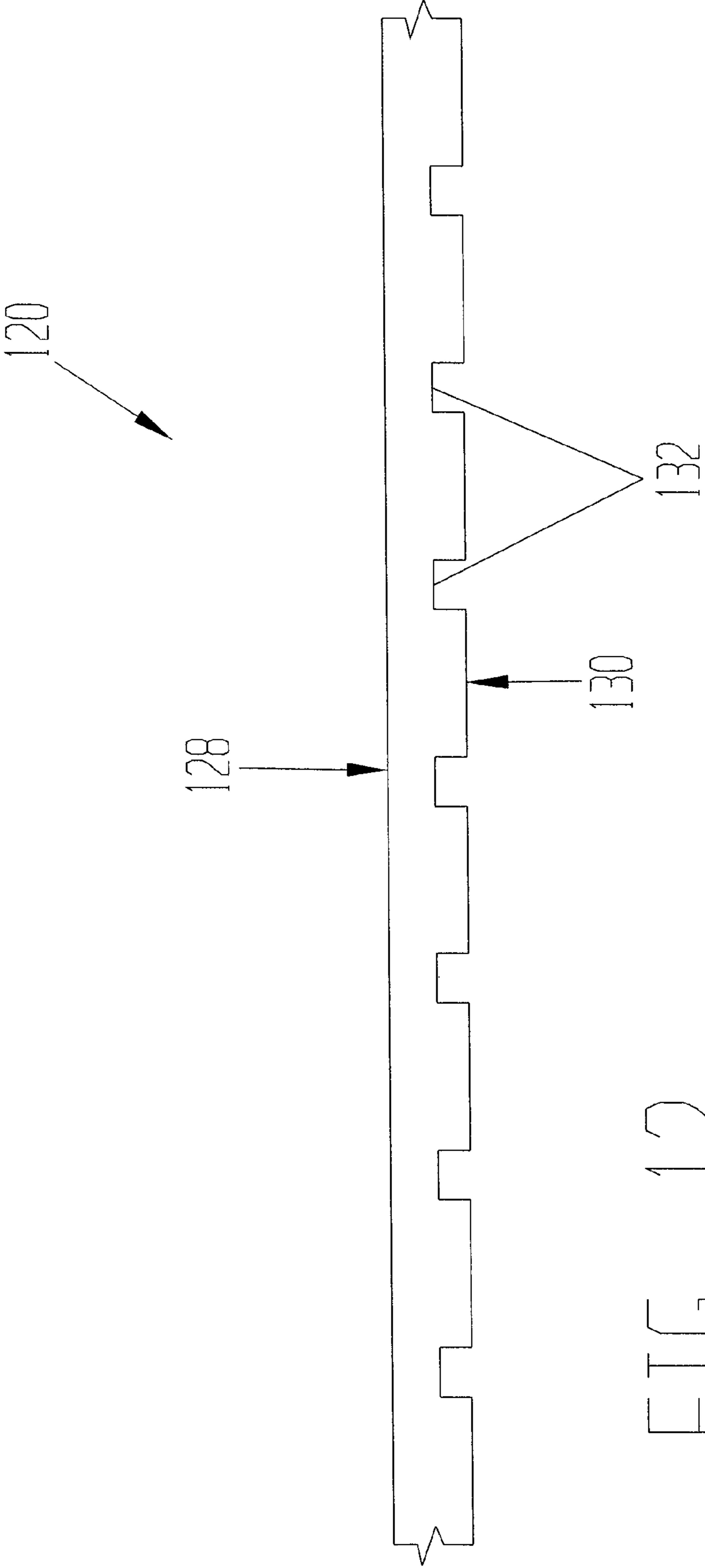
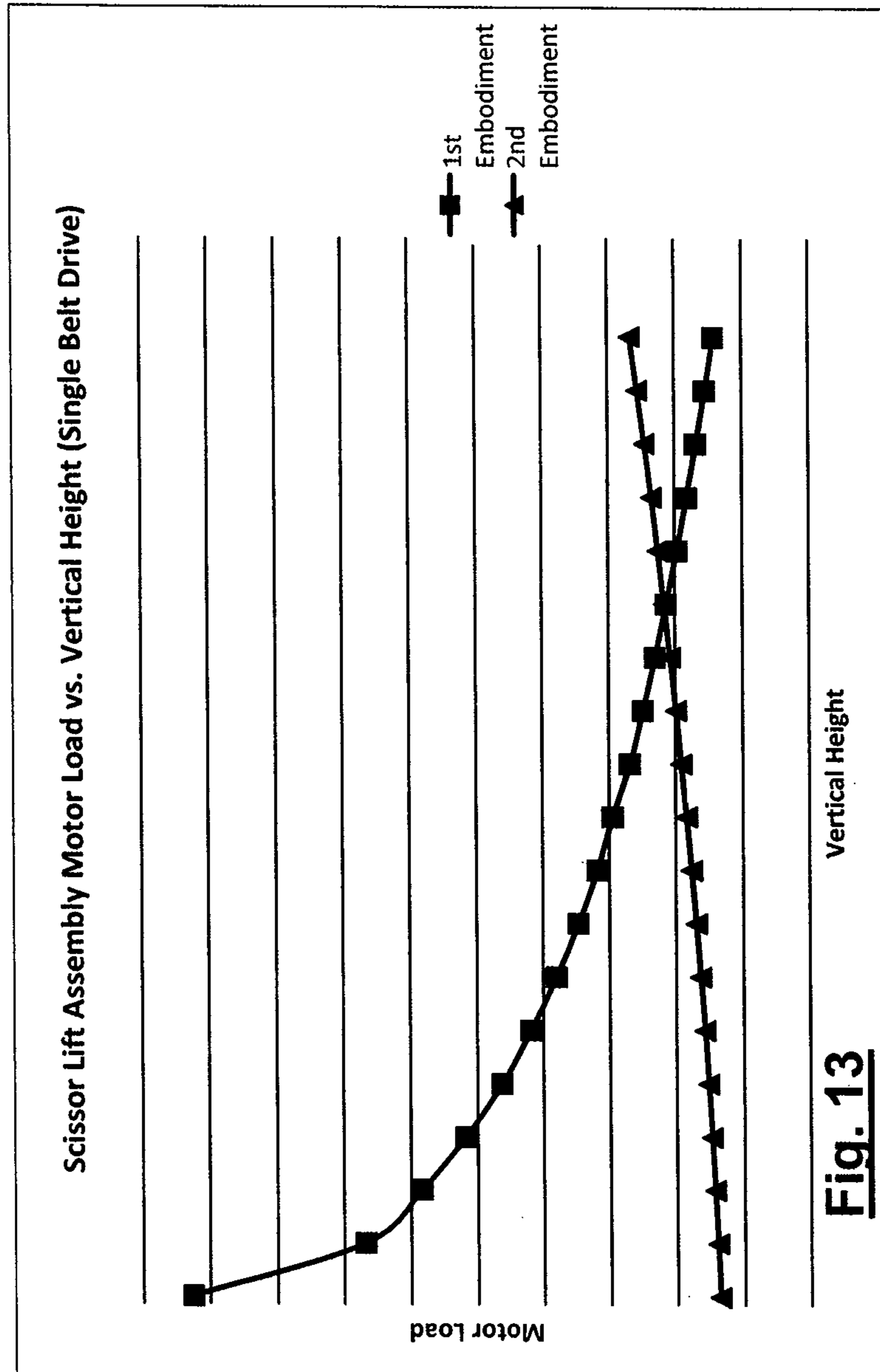


FIG. 12



**BELT-DRIVEN TRANSPORTATION SYSTEM**

## BACKGROUND OF THE INVENTION

## 1. Field of the Invention

The present invention relates generally to belt-driven transportation systems, and particularly to such systems configured as lift assemblies and conveyor assemblies.

## 2. Description of the Related Art

Transportation systems such as lift assemblies are well known for moving items or people between two vertically differing locations. Transportation systems such as conveyor assemblies are well known for moving items or people between two horizontally differing locations. It is also known to move the platform or carriage of these systems through a belt-driven apparatus.

One well-known and useful type of lift assembly is the scissor-type, which moves between a lowered or contracted state, and an elevated or extended state as its load-supporting platform is moved between differing vertical locations. Such lift assemblies are commonly driven through hydraulic cylinders, screw-drive mechanisms or expandable pneumatic bladder arrangements, and some prior types of scissor lift assemblies are belt-driven. These prior lift assemblies, however, often require a substantial amount of power, particularly when moving from their lowermost contracted states, or are difficult to reliably, precisely control.

Further, some prior belt-driven transportation systems can be problematic to install or repair, and sometimes to operate, due to the belt being moved out of its guided position along its designed path over pulleys that it engages. Another problem with some types of belt-driven transportation systems is that they rely on traction between the belt and the pulleys to operate, which can lead to slippage under heavy loading and result in unintentional lift collapse. Additionally, some transportation systems of the types described above, particularly lift assemblies, undesirably require operating space that cannot be easily accommodated or interferes with carrying out the operation to which the system is applied. For example, some prior lift assemblies have platform heights in their fully contracted states that require the load to first be lifted a substantial vertical distance from the level of a floor, on which the base is positioned, to place it on the platform. Thus, it would be preferable to minimize the height of the platform in its fully contracted or lowered state.

A transportation system configured as a lift assembly or conveyor assembly that addresses at least one of the above-mentioned problems is desirable.

## SUMMARY OF THE INVENTION

The present invention provides a scissors-type lift assembly including a base and a platform, the lift assembly having elevated and lowered states in which the platform and base are distant and proximate, respectively. The lift assembly further includes a pair of first and second scissor arms each having upper and lower ends respectively coupled to the platform and the base, the pair of scissor arms pivotably connected to each other intermediate their respective upper and lower ends about a central pivot axis. Also provided are a first pulley arrangement that includes at least one first pulley disposed about a first pulley axis, and a second pulley arrangement that includes at least one second pulley disposed about a second pulley axis, the first and second pulley axes disposed in a pulley plane and having lateral movement relative to each other as the lift assembly is moved between its elevated and lowered states, the pulley plane fixed relative to the base. A

spool is rotatable about an axis fixed to the base, and a unitary belt is guided through a path defined by the first and second pulleys. The belt has opposed first and second ends, the belt first end engaged with the spool onto which the belt is wound and from which the belt is unwound. The first and second pulley axes have a first lateral distance therebetween in the lift assembly elevated state and a second lateral distance therebetween greater than the first distance in the lift assembly lowered state, and the first and second pulley axes pulled laterally together by the belt as it is wound onto the spool.

The invention further provides a scissors-type lift assembly including a base and a platform, the lift assembly having extended and contracted states with the platform at a relatively greater distance from the base in the extended state than in the contracted state. The lift assembly further includes a pair of first and second elongate arms each having opposite ends and extending between the base and the platform. The first and second arms are pivotably connected together and cross each other substantially at a first angle in the extended state and substantially at a second angle in the contracted state. Also included are a first pulley arrangement including at least one first pulley rotatably disposed about a first pulley axis, and a second pulley arrangement including at least one second pulley rotatably disposed about a second pulley axis, the first and second pulley axes defining a stationary plane in which the first and second pulley axes have relative lateral movement. One of the first and second pulley arrangements is affixed to one of the first and second elongate arms, and the arms are moved between the first and second angles with relative lateral movement of the pulley axes. A spool is rotatable about an axis of rotation fixed relative to the base, and a unitary belt is guided through a path defined by the first and second pulleys, the belt having opposed first and second ends. The belt first end is engaged with the spool onto which the belt is wound and from which the belt is unwound, and the first and second pulley axes are moved laterally in opposite directions relative to each other as the belt is respectively wound onto and unwound from the spool.

The present invention further provides a lift assembly including a base, a platform, and a pair of first and second elongate scissor arms each having a first end coupled to the base and a second end coupled to the platform, the first and second scissor arms pivotably connected to each other at a location between their respective first and second ends. The lift assembly further has a belt and pulley apparatus that includes a unitary belt having opposite first and second ends, a belt spool rotatably attached to the base and having reversible rotation about a spool axis. The belt first end is secured to the spool, and the belt second end is secured to the base. The belt is respectively wound onto and unwound from the spool with reversing rotation of the spool. Also included are a first pulley arrangement having a plurality of first pulleys disposed about a first axis, and a second pulley arrangement having at least one second pulley disposed about a second axis fixed relative to the base. The first and second axes are substantially parallel with each other and laterally movable relative to each other in a stationary plane, with the first axis fixed to one of the first and second scissor arms.

The present invention further includes a belt-driven transportation system including a first set of pulleys rotatably attached to a first member and a second set of pulleys rotatably attached to a second member. The first and second members have relative movement to each other. The system further includes a unitary belt that is guided through a path defined by the first and second sets of pulleys with the path substantially lying in a stationary plane, and a plurality of retention rollers. A retention roller is disposed adjacent each pulley, with the



belt extending about the pulley and between the pulley and the retention roller. The positioning of the belt substantially on the path is maintained by the plurality of retention rollers, and the first and second members are moved relative to each other with changes in the length of the path.

There has thus been outlined, rather broadly, certain features of embodiments of the invention in order that the detailed descriptions thereof may be better understood, and in order that the present contribution to the art may be better appreciated. Additional or alternative features of embodiments of the invention are described in further detail below.

In this respect, before explaining embodiments of the invention in detail, it is to be understood that the invention is not limited in its application to the details of construction and to the arrangements of the components set forth in the following description or illustrated in the drawings. The invention is capable of other embodiments and of being practiced and carried out in various ways. Also, it is to be understood that the phraseology and terminology employed herein are for the purpose of description and should not be regarded as limiting.

To accomplish the above and related objects, the invention may be embodied in the forms illustrated in the accompanying drawings, attention being called to the fact, however, that the drawings are illustrative only, and that changes may be made in the specific constructions illustrated. Moreover, it is to be noted that the accompanying drawings are not necessarily drawn to scale or to the same scale. In particular, the scale of some of the elements of the drawings may be exaggerated to emphasize characteristics of the elements.

#### BRIEF DESCRIPTION OF THE DRAWINGS

Various other objects, features and attendant advantages of the present invention will become fully appreciated as the same becomes better understood when considered in conjunction with the accompanying drawings, in which like reference characters designate the same or similar parts throughout the several views, and wherein:

FIG. 1 is a perspective view of a scissor-type lift assembly according to a first embodiment in an elevated state;

FIG. 2 is a front view of the lift assembly shown in FIG. 1 in a lowered state;

FIG. 3 is a side view of the lift assembly shown in FIG. 1 in an elevated state;

FIG. 4 is a top view of the lift assembly shown in FIG. 1 in a lowered state;

FIG. 5 is a top view of the lift assembly shown in FIG. 1 in a lowered state with its platform removed from view;

FIG. 6 is a perspective view of a scissor-type lift assembly according to a second embodiment in an elevated state;

FIG. 7 is a side view of the lift assembly shown in FIG. 6 in an elevated state;

FIG. 8 is a side view of the lift assembly shown in FIG. 6 in a lowered state;

FIG. 9 is a side view of the lift assembly shown in FIG. 6 in an elevated state, with the second pair of scissor arms removed from view;

FIG. 10 is a perspective view of a conveyor or lift assembly according to a third embodiment, with its platform shown in phantom lines;

FIG. 11 is a perspective view of a variant of the conveyor or lift assembly shown in FIG. 10 with its platform, base and guide structure removed from view;

FIG. 12 is a fragmented side view of a unitary belt used with any of the first, second and third embodiments; and

FIG. 13 is a fragmented, partially sectioned view of an exemplary pulley, retainer roller and unitary belt of any of the first, second and third embodiments; and

FIG. 13 is a graph comparing the motor loads of the first and second embodiment scissor lift assemblies (each with single belt drive) as they respectively move from their lowered to their elevated states.

While the invention is susceptible to various modifications and alternative forms, specific embodiments thereof are shown by way of example in the drawings and may herein be described in detail. It should be understood, however, that the drawings and detailed description thereto are not intended to limit the invention to the particular forms disclosed, but on the contrary, the intention is to cover all modifications, equivalents and alternatives falling within the spirit and scope of the present invention as defined by the appended claims.

#### DETAILED DESCRIPTION

Referring FIGS. 1-5 there is shown first embodiment scissor-type lift assembly 20 which has a base 22 and a platform 24, each of which may be made of steel. Platform 24 has upwardly facing supporting surfaces 25. Lift assembly 20 has an elevated state 26 in which the platform and base are distant from each other, and a lowered state 28 in which the lift assembly base and platform are proximal to each other. FIGS. 1 and 3 show lift assembly 20 in elevated state 26, and FIGS. 2, 4 and 5 show lift assembly 20 in lowered state 28.

Disposed between and operatively connected to base 22 and platform 24 is scissor arm assembly 30 which includes first pair of scissor arms 32 and second pair of scissor arms 34. Between the first and second pair of scissor arms is brace 36, which extends between first arm 38 of each of the first and second pairs of scissor arms 32, 34. The first and second pairs of scissor arms 32 and 34 each further includes second arm 40. Arms 38, 40 may be made of steel. Upper end 42 of each first and second arm 38, 40 engages platform 24, and lower end 44 of each first and second arm 38, 40 engages base 22. Respective to each pair of scissor arms 32, 34, first and second arms 38, 40 are pivotably connected to each other through a bolted connection 46 about pivot axis 48 of scissor arm assembly 30.

Each first arm 38 has a bolted connection 50 to platform 24, which defines pivot axis 51, and each second arm 40 has a bolted connection 52 to base 22, which defines pivot axis 53. Base 22 includes guide tracks 54 in which lower ends 44 of the first arms 38 are slidably engaged, and platform 24 includes guide tracks 56 in which upper ends 42 of second arms 40 are slidably engaged.

Lift assembly 20 further includes first pulley arrangement 58 which is connected to brace 36 with bolts 60. First pulley arrangement 58 includes a plurality of pulleys 62. Five pulleys 62, identified as pulleys 62a through 62e in FIG. 5, are independently rotatable about shaft 64, which extends between and is fixed to laterally spaced blocks or support members 66 and 68, into which bolts 60 are threadedly received. Blocks 66, 68 may be made of steel.

Adjacent pulleys 62 may have therebetween a bushing or other friction reducing member (not shown) to facilitate their independent rotation relative to each other about shaft 64. Each of pulleys 62 has opposing flanges 70 between which is defined a belt engaging circumferential pulley surface 330 discussed further below.

First pulley arrangement 58 further includes shaft 72 that is parallel with shaft 64, and about which are disposed independently rotatable retainer rollers 74, one for each of pulleys 62. Shaft 72 also extends between and is fixed to laterally spaced

blocks or support members **66** and **68**. Thus, pulleys **62** and retainer rollers **74** are respectively rotatable about parallel axes of rotation **76** and **78** respectively defined by shafts **64** and **72**. It is thus understood that first pulley arrangement **58** moves relative to base **22** with first scissor arms **38** as lift assembly **20** is moved between its elevated and lowered states **26**, **28**.

Base **22** includes laterally extending cross braces **80**, **82**, and **84** which extend between and are fixed to its opposite rails **85**, **86**. Attached to cross brace **82** is second pulley arrangement **88**, which is fixed to base **22** by means of bolts **90**. Second pulley arrangement **88** includes a plurality of pulleys **92**. Four pulleys **92**, identified as pulleys **92a** through **92d** in FIG. **5**, are independently rotatable about shaft **94**, which extends between and is fixed to laterally spaced blocks or support members **96** and **98**, into which bolts **90** are threadedly received.

Adjacent pulleys **92** may have therebetween a bushing or other friction reducing member (not shown) to facilitate their independent rotation relative to each other about shaft **94**. Each of pulleys **92** has opposing flanges **100** between which is defined a belt engaging circumferential pulley surface **330** discussed further below.

Second pulley arrangement **88** further includes shaft **102** that is parallel with shaft **94**, and about which are disposed independently rotatable retainer rollers **104**, one for each of pulleys **92**. Shaft **102** also extends between and is fixed to laterally spaced blocks or support members **96** and **98**. Thus, pulleys **92** and retainer rollers **104** are respectively rotatable about parallel axes of rotation **106** and **108** respectively defined by shafts **94** and **102**.

Lift assembly **20** further includes motor drive assembly **112**, which includes reversible servo or stepper motor **114**, which is in driving, co-axial engagement with spool or pulley **116**, which has opposed flanges **118** between which is defined a belt engaging portion. Unitary belt **120** having opposite first and second ends **121**, **122** is guided along a path defined by pulleys **62** and **92**, and first end **121** of belt **120** is connected to spool **116** such that rotation of the spool about its axis of rotation **124** will either wind belt **120** onto the spool or unwind belt **120** from the spool. Spool axis of rotation **124** is substantially parallel with pulley axes **76** and **106**.

A plane **126** is defined by pulley axes **76** and **106**, and maintains a substantially horizontal orientation, parallel with platform surfaces **25**. As best seen in FIG. **3** plane **126** is disposed within the height defined by base **22**. The first and second pulley arrangements **58**, **88** move toward and away from each other in plane **126** as lift assembly **20** is moved between its elevated and lowered states **26**, **28**. Relative to lift assembly **20**, plane **126** is stationary.

Referring now to FIG. **12**, it can be seen that belt **120** is substantially flat and has an outer side **128** and an opposite inner side **130**. Inner side **130** is provided with a plurality of longitudinally spaced and laterally extending grooves **132** which define ribs on the inner side **130** of the belt, thereby facilitating the belt's ability to wrap itself about the pulleys and the spool. Belt **120** can be any of a number of commercially available belts of suitable width, length and material properties that may be selected based on the requirements of the application for which lift assembly **20** is to be used.

Referring to FIG. **5**, elongate unitary belt **120** extends from about spool **116** rightwardly to first pulley arrangement **58** where it is wrapped about pulley **62a**, and from pulley **62a** belt **120** reverses course and at a slight return angle returns toward second pulley arrangement **88** where it is then wrapped about pulley **92a**. From pulley **92a** belt **120** reverses course and at a slight return angle returns toward first pulley

arrangement **58** where it is wrapped about pulley **62b**. From pulley **62b** belt **120** reverses course and at a slight return angle returns toward second pulley arrangement **88** where it is wrapped about pulley **92b**. From pulley **92b** belt **120** reverses course and at a slight return angle returns toward first pulley arrangement **58** where it is wrapped about pulley **62c**. From pulley **62c** belt **120** reverses course and at a slight return angle returns toward second pulley arrangement **88** where it is wrapped about pulley **92c**. From pulley **92c** belt **120** reverses course and at a slight return angle returns toward first pulley arrangement **58** where it is wrapped about pulley **62d**. From pulley **62d** belt **120** reverses course and at a slight return angle returns toward second pulley arrangement **88** where it is wrapped about pulley **92d**. From pulley **92d** belt **120** returns in a direction toward the first pulley arrangement **58** and wraps about pulley **62e**, and reverses course returning toward second pulley arrangement **88** and its second end **122** is affixed to cross brace **82** of base **22**. The return angle along which belt **120** extends between the first and second pulley arrangements **58**, **88** is relative to their parallel pulley axes **76**, **106**, and will vary slightly as lift assembly **20** moves between its elevated state **26** in which the return angle is at a maximum, and its lowered state **28** in which the return angle is at a minimum. The belt's ability to accommodate the return angles is facilitated in part by the belt engaging circumferential pulley surface **330** of pulleys **62** and **92**.

One of ordinary skill in the art will recognize that first pulley arrangement **58**, which is attached by its blocks **66** and **68** to scissor arm assembly **30**, is biased rightwardly as shown in FIG. **5** and away from second pulley arrangement **88** which is affixed to base **22**, to maintain an amount of tension on belt **120**. The first and second pulley arrangements **58**, **88** are biased apart from each other under the weight of scissor arm assembly **30**, platform **24**, and any load on the platform. Thus, axes **76**, **106** are biased laterally apart from each other in plane **126**. Pulleys **92** and pulley **62** may be of a common diameter, and thus the lengths of belt **120** extending therebetween on opposite sides of plane **126** respectively lay in common planes parallel to plane **126**.

As belt **120** is wound onto spool **116**, the length of the path along which belt **120** is guided about pulley **62** and **92** becomes shortened, and axes **106** and **76** are moved laterally within plane **126** towards one another as the lower ends **44** of first arms **38** are moved leftwardly towards lower ends **44** of second arms **44**, thereby changing the angle at which the elongate first and second arms **38**, **40** are crossed, and platform **24** is moved upwardly towards its elevated state **26**.

Conversely, as belt **120** is unwound from about spool **116**, the weight of scissor arm assembly **30**, platform **24**, and any load it bears, urges scissor arm assembly **30** towards a collapsed state, in which the angle at which the elongate first and second scissor arms **38**, **40** are crossed changes such that first pulley arrangement **58** is moved rightwardly away from second pulley arrangement **88**, and lift assembly **20** is moved towards its lowered state **28**.

Referring now to FIGS. **6-9**, there is shown second embodiment scissor-type lift assembly **20A**, which is similar in general configuration to first embodiment scissor lift assembly **20** except as otherwise described below.

Scissor lift assembly **20A** includes a motor drive assembly **112** in which reversible servo or stepper motor **114** is oriented along a drive axis **134** that is substantially perpendicular to spool drive axis **124**, the output shaft of motor **114** extending into a gear drive unit in which the spool driving output shaft that defines axis **124** extends laterally in opposite directions from the gear drive housing. Each laterally extending end of the output shaft has a spool **116** affixed thereto. As best shown

in FIG. 6, lift assembly 20A includes a pair of first and second drive belts 120a, 120b which work in parallel and provides a measure of safety beyond the single belt arrangement shown in first embodiment lift assembly 20. The depicted dual-belt configuration of lift assembly 20A protects the user and the lifted load from injury or damage due to failure of one of the drive belts 120a and 120b. It is to be understood that first embodiment lift assembly 20 may be outfitted with a dual belt drive like that shown in second embodiment scissor lift assembly 20A. Conversely, second embodiment scissor lift assembly 20A may be outfitted with a single belt drive like that shown in first embodiment lift assembly 20.

In second embodiment scissor lift assembly 20A, the first and second pulley arrangements 58, 88 are disposed between the first and second scissor arms 38, 40 of the first and second pairs of scissor arms 32, 34. First pulley arrangement 58 includes a pulley arrangement 58a associated with first drive belt 120a and another pulley arrangement 58b associated with second drive belt 120b. The pulleys 62 of each first pulley arrangement 58a and 58b are independently rotatably disposed on a common shaft 64 extending between and fixed to laterally spaced blocks or support members 66 and 68. Shaft 64 defines axis 76 about which pulleys 62 rotate. At each of the opposite ends of shaft 64, between the laterally outermost pulleys 62 and the adjacent block 66, 68, there is provided a cam follower or roller rotatable about axis 96. Cam followers 140 and 142 respectively engage the rightwardly-facing edges of the first and second pairs of scissor arms 32 and 34, as shown in FIGS. 6 and 7. Each cam follower 140, 142 includes two adjacent, relatively rotatable roller elements 156, 158.

Similarly, second pulley arrangement 88 includes a pulley arrangement 88a associated with first drive belt 120a and another pulley arrangement 88b associated with second drive belt 120b. The pulleys 92 of each second pulley arrangement 88a and 88b are independently rotatably disposed on a common shaft 94 extending between and fixed to laterally spaced blocks or support members 96 and 98. Shaft 94 defines axis 106 about which pulleys 92 rotate. At each of the opposite ends of shaft 94, between the laterally outermost pulleys 92 and the adjacent block 96, 98, there is provided a cam follower or roller rotatable about axis 106. Cam followers 144 and 146 respectively engage the leftwardly-facing edges of the first and second pairs of scissor arms 32 and 34, as shown in FIGS. 6 and 7. Like cam followers 140, 142, each cam follower 144, 146 includes two adjacent, relatively rotatable roller elements 156, 158.

In each of cam followers 140, 142, 144, 146, roller element 156 is located laterally inboard of roller element 158 and is in rolling contact with an edge of a second scissor arm 40, and laterally outboard roller element 158 is in rolling contact with an edge of a first scissor arm 38. Thus, the roller elements 156, 158 of each cam follower 140, 142, 144, 146 rotate in opposite directions as they roll along their respective scissor arm edges.

Each inboard roller element 156 of a cam follower 140, 142, 144, 146 engages a cam surface 150 defined on an edge of a second scissor arm 40 of the first and second pairs of scissor arms 32, 34. Similarly, each outboard roller element 158 of a cam follower 140, 142, 144, 146 engages a cam surface 148 defined on an edge of a first scissor arm 38 of the first and second pairs of scissor arms 32, 34.

As perhaps best shown in FIG. 9, axes of rotation 76 and 106 of the first and second pulley arrangements 58, 88 lie in a plane 126, the orientation of which is maintained throughout movement of the lift assembly 20A between its elevated and lowered states. In lift assembly 20A, plane 126 remains sub-

stantially horizontal, and also includes pivot axis 48. Here, as opposed to first embodiment lift assembly 20, plane 126 is not fixed relative to base 22, but rather moves vertically up and down with movement of lift assembly 20A between its elevated and lowered states 26, 28.

The tension on belts 120a and 120b maintain cam followers 140, 142 and 144, 146 of the first and second pulley arrangements 58 and 88 in engagement with surfaces 150 and 148 of first and second arms 38 and 40 of the first and second pairs of scissor arms 32, 34. Cam surfaces 148, 150 are designed to easily facilitate an initial movement of lift assembly 20A from its lowered state 28 to minimize additional loading on motor 114 than would otherwise be required in the absence of the cam surfaces being so designed. As shown, the cam surfaces 148, 150 establish, in the lowered state 28, a gradual slope along which roller elements 56, 58 are forced to roll in response to belts 120a, 120b being wound onto their respective spools 116. The slope increases somewhat as lift assembly 20A is moved out of its lowered state 28 and axes 96, 106 are pulled laterally toward each other in plane 126. Once the lift assembly 20A has been moved from its initial, lowered state 28, the cam surfaces 148, 150 engaged by followers 140, 142, 144, 146 change their slopes such that the load on motor 114 is maintained at an approximately constant level, and an increased rate of change in height of platform 24 results.

FIG. 13 provides a qualitative comparison between a first embodiment scissor lift assembly (e.g., lift assembly 20) and an comparably sized second embodiment scissor lift assembly (e.g., lift assembly 20A), the latter having cam surfaces 148, 150 substantially configured as shown in FIGS. 6-9 and described above. The first and second embodiment scissor lift assemblies compared through the plotted curves of FIG. 13 each have a single belt drive, as described above, and a common number of first and second pulleys 62, 92 in their respective first and second pulley arrangements 58, 88. Additionally, the comparison is through a range between lowered and elevated states that are respectively established at common platform heights, and the loads borne by their respective platforms are identical.

The motor load is derived as a function of the tension force in the segment of belt 120 extending from about spool 116. FIG. 13 shows, in raising the platform from the lift assembly lowered or contracted state 28 (at the left side of the graph) to the lift assembly elevated or extended state 26 (at the right side of the graph) the second embodiment scissor lift substantially reduces the motor loading vis-à-vis the first embodiment lift assembly up to a height substantially past the midpoint of platform travel, where the two lines intersect. After this height is reached, the motor load of the second embodiment lift assembly is comparatively higher, but not substantially so. Indeed, the second embodiment lift assembly motor load has only gradual, substantially linear increases as it moves from the lowered to the elevated state. On the other hand, the first embodiment lift assembly shows a dramatic initial reduction in motor loading immediately after moving from its lowered state, with further, more gradual and substantially linear motor load reduction occurring in the latter portions of upward platform travel towards the elevated state.

Clearly, it would be desirable to avoid the initial, comparatively much higher motor loading in moving from the lift assembly lowered state as the second embodiment lift assembly allows, for the substantially greater initial loading comes with attendant increases in energy use and stresses on the belt and other lift assembly components. However, it is contemplated that a second embodiment scissor lift assembly will likely have greater cost than a comparable first embodiment

scissor lift assembly. On the other hand, the relatively flatter motor load curve of the second embodiment scissor lift assembly will likely reduce the need for an expensive motor control system.

Those of ordinary skill in the art will recognize that, in either of the first and second embodiment scissor lift assemblies, motor loading can be reduced by correspondingly increasing the number of first and second pulleys **62, 92** of the first and second pulley arrangements **58, 88** over which belt **120** is wrapped, with belt **120** being correspondingly lengthened to accommodate its increased path. This reduction in motor loading/belt tension would, however, result in comparatively slower travel between the lift assembly contracted and elevated states.

Returning to FIGS. **6-9**, the cam surfaces **148** of second embodiment scissor lift assembly **20A** are each contoured near opposite ends **42, 44** of the first arms **38** to define recesses **152**. Similarly, cam surfaces **150** are each contoured near opposite ends **42, 44** of the second arms **40** to define recesses **154**. In the lowered state **28** of lift assembly **20A**, as shown in FIG. **8**, the cam followers **140, 142, 144, 146** are received in spaces defined by recesses **152** and **154** to minimize the total height of lift assembly **20A** between the bottom most part of its base **22** and load supporting surfaces **25** of platform **24** in lowered state **28**.

First and second pulley arrangements **58** and **88** of second embodiment lift assembly **20A** respectively further include shafts **72, 102**, that are fixed to and extend in parallel with shafts **64, 94** between laterally spaced blocks or support members **66, 68** and **96, 98**. Blocks **66, 68, 96, 98** are preferably structured and interconnected to maintain a consistent orientation of axes **78** and **108** relative to plane **126**. They may, for example, both lie in plane **126** as shown. Between the first and second pulley arrangements **58, 88**, a pair of blocks (e.g. **66** and **96**, or **68** and **98**) may be linked together laterally outside the adjacent pair of scissor arms **32** or **34**, for example.

Referring now to FIGS. **10** and **11** there is shown a conveyor or lift assembly **220**, a third embodiment of the present invention. Assembly **220** includes base **222**, and as a conveyor assembly may also include platform **224** (shown in phantom lines in FIG. **10**) defining support surface **225**. Conveyor or lift assembly **220** has a centered state **226** and is arranged to provide conveying or lifting movement longitudinally in the directions indicated by double-headed arrow **228**. A load supported by carriage **230** is thus moved longitudinally through operation of assembly **220**. The load may be supported by platform **224** and moved substantially horizontally as a conveyor or, alternatively, a load supported by carriage **230** may be moved substantially vertically, as a lift or elevator.

Carriage **230**, to which platform **224** may be attached, is disposed between a pair of longitudinally extending rails **232** and **234**. Carriage **230** is provided with bearing elements **236** that are received in and supported by guide tracks **238, 240** of rails **232, 234**. Bearing elements **236** support the load placed on surface **225** of platform **224** when assembly **220** is in a substantially horizontal orientation, or otherwise constrain the movement of carriage **230** away from base **222**. Base **222**, platform **224**, and rails **232, 234** may be made of steel.

In FIGS. **10** and **11**, the longitudinal direction is that indicated by arrow **228**, the lateral direction being substantially perpendicular thereto in the directions of carriage pulley axis **242** defined by shaft **243**, carriage pulley axis **244** defined by shaft **245**, and carriage retainer roller axis **246** defined by shaft **247**. Shafts **243, 245**, and **247** extend between and are fixed to a pair of laterally spaced blocks **248, 250** to which bearings **236** are attached at the laterally outward sides

thereof. Referring to FIG. **11**, a variation of assembly **220** is shown that includes a pair of retainer roller axes **246a** and **246b** associated with a pair of shafts **247a** and **247b** extending between and fixed to blocks **248, 250**, instead of the single shaft **247** and axis **246** shown in FIG. **10**. Platform **224** may be attached to blocks **248, 250**.

Respectively disposed about shafts **243** and **245** are like-numbered pluralities of independently rotatable carriage pulleys **252** and **254** which may be identical to above-discussed pulleys **62, 92**. Disposed about shaft **247** (or about shafts **247a** and **247b**) are a plurality of retention rollers **256** which may be identical to above-discussed retention rollers **74, 104**. In FIG. **10**, where a single shaft **247** carries retention rollers **256**, there are a like number of retention rollers **256** and pairs of carriage pulleys **252** and **254**, each retention roller disposed between and having a common relationship with each pair of carriage pulleys. In the variation shown in FIG. **11**, there is a retention roller **256** associated with each one of carriage pulley **252, 254**. The relationship between each pulley and its retention roller is as discussed above.

Assembly **220** further includes first pulley arrangement **258**, shown on the right side of FIGS. **10** and **11**. First pulley arrangement **258** includes shaft **264** defining axis **276** about which is disposed a plurality of independently rotatable pulleys **62**. Shaft **264** extends between and is fixed to a laterally spaced pair of blocks **266, 268** which are affixed to base **222**. First pulley arrangement **258** further includes shaft **272** defining axis **278**. Shaft **272** extends between and is fixed to blocks **266, 268**, and is parallel to shaft **264**. Disposed about shaft **272** is a plurality of independently rotatable retention rollers **74**, one for each pulley **62**.

Referring to the left side of FIGS. **10** and **11**, assembly **220** further includes a second pulley arrangement **288** which is similar in configuration to first embodiment pulley assembly **258**. Second pulley arrangement **288** includes shaft **294** defining axis **306** about which is disposed a plurality of independently rotatable pulleys **92**, with shaft **294** extending between and fixed to laterally spaced blocks **296, 298** which are affixed to base **222**. Second pulley arrangement **288** further includes shaft **302** defining axis **308**. Shaft **302** extends between and is fixed to blocks **296, 298**, and is parallel to shaft **294**. Disposed about shaft **302** is a plurality of independently rotatable retention rollers **104**, one for each pulley **92**. Blocks **248, 250, 266, 266, 296, and 298** may be made of steel.

Extending between first pulley arrangement **258** and carriage **230** is first drive belt **120a**, which is guided over a path defined by pulleys **62** and **252**. As shown, there are four pulleys **62** in first pulley arrangement **258**, identified as **62a-d**, and five carriage pulleys **252**, identified as carriage pulleys **252a-e**. Elongate unitary belt **120a** extends from about spool **116a** leftwardly to carriage **230** where it is wrapped about carriage pulley **252a**, and from pulley **252a** belt **120a** reverses course and at a slight return angle returns toward first pulley arrangement **258** where it is then wrapped about pulley **62a**. From pulley **62a** belt **120a** reverses course and at a slight return angle returns toward carriage **230** where it is wrapped about pulley **252b**. From pulley **252b** belt **120a** reverses course and at a slight return angle returns toward first pulley arrangement **258** where it is wrapped about pulley **62b**. From pulley **62b** belt **120a** reverses course and at a slight return angle returns toward carriage **230** where it is wrapped about pulley **252c**. From pulley **252c** belt **120a** reverses course and at a slight return angle returns toward first pulley arrangement **258** where it is wrapped about pulley **62c**. From pulley **62c** belt **120a** reverses course and at a slight return angle returns toward carriage **230** where it is wrapped about pulley **252d**. From pulley **252d** belt **120a** reverses course and at a slight

return angle returns toward first pulley arrangement 258 where it is wrapped about pulley 62d. From pulley 62d belt 120a returns in a direction toward carriage 230 and wraps about pulley 252e, and reverses course returning toward first pulley arrangement 258 and its second end 122 is affixed to base 222. The return angle along which belt 120a extends between the carriage 230 and the first pulley arrangement 258 is relative to their parallel pulley axes 242, 276 and will vary slightly as carriage 230 moves longitudinally in the directions indicated by arrow 228, the return angle being at a maximum when carriage 230 is rightmost and being at a minimum when carriage 230 is leftmost, as viewed in FIGS. 10 and 11. The belt's ability to accommodate the return angles is facilitated in part by the belt engaging circumferential pulley surface 330 of pulleys 62 and 252.

Extending between second pulley arrangement 288 and carriage 230 is second drive belt 120b, which is guided over a path defined by pulleys 92 and 254. As shown, there are four pulleys 92 in second pulley arrangement 288, identified as 92a-d, and five carriage pulleys 254, identified as carriage pulleys 254a-e. Elongate unitary belt 120b extends from about spool 116b rightwardly to carriage 230 where it is wrapped about carriage pulley 254a, and from pulley 254a belt 120b reverses course and at a slight return angle returns toward second pulley arrangement 288 where it is then wrapped about pulley 92a. From pulley 92a belt 120b reverses course and at a slight return angle returns toward carriage 230 where it is wrapped about pulley 254b. From pulley 254b belt 120b reverses course and at a slight return angle returns toward second pulley arrangement 288 where it is wrapped about pulley 92b. From pulley 92b belt 120b reverses course and at a slight return angle returns toward carriage 230 where it is wrapped about pulley 254c. From pulley 254c belt 120b reverses course and at a slight return angle returns toward second pulley arrangement 288 where it is wrapped about pulley 92c. From pulley 92c belt 120b reverses course and at a slight return angle returns toward carriage 230 where it is wrapped about pulley 254d. From pulley 254d belt 120b reverses course and at a slight return angle returns toward second pulley arrangement 288 where it is wrapped about pulley 92d. From pulley 92d belt 120b returns in a direction toward carriage 230 and wraps about pulley 254e, and reverses course returning toward second pulley arrangement 288 and its second end 122 is affixed to base 222. The return angle along which belt 120b extends between the carriage 230 and the second pulley arrangement 288 is relative to their parallel pulley axes 244, 306 and will vary slightly as carriage 230 moves longitudinally in the directions indicated by arrow 228, the return angle being at a maximum when carriage 230 is leftmost and being at a minimum when carriage 230 is rightmost, as viewed in FIGS. 10 and 11. The belt's ability to accommodate the return angles is facilitated in part by the belt engaging circumferential pulley surface 330 of pulleys 92 and 254.

As described above, first end 121 of each belt 120 is attached to its respective spool 116 and is wound onto or unwound from the spool 116, which changes the length of the path over which the belt 120 extends, the path being defined by the respective pulleys over which it is wrapped. The second end 122 of each belt 120 is affixed to base 222 adjacent the respective first or second pulley arrangement.

Assembly 220 includes a first motor drive assembly 312a which is arranged for pulling belt 120a rightward through rotation of its motor 314a which may be rotatable in only a single direction. As shown, the direction of rotation of motor 314a of motor drive assembly 312a when pulling carriage 230 rightwardly as belt 120a is wound onto spool 116a, is

clockwise. In motor drive assembly 312a, the output shaft of motor 314a may extend through a clutch housing 318a that includes a one way clutch which allows belt 120a to be unwound from spool 116a as carriage 230 is pulled leftwardly by second motor drive assembly 312b when motor 314a is de-energized.

Similarly, second motor drive 312b of assembly 220 is arranged for pulling belt 120b leftwardly through rotation of its motor 314b which may be rotatable in only a single direction. As shown, the direction of rotation of motor 314b of motor drive assembly 312b when pulling carriage 230 leftwardly as belt 120b is wound onto spool 116b, is counterclockwise. In motor drive assembly 312b, the output shaft of motor 314b may extend through a clutch housing 318b that includes a one way clutch which allows belt 120b to be unwound from spool 116b as carriage 230 is pulled rightwardly by first motor drive assembly 312a when motor 314b is de-energized.

One of ordinary skill in the art will recognize that motor drive assemblies 312a and 312b are individually and exclusively energized to move a load supported by carriage 230 in one of the two directions indicated by arrow 228. Movement of carriage 230 rightward as shown in FIGS. 10 and 11, for example, involves energizing motor drive assembly 312a which will wind belt 120a onto spool 116a, shortening the path over which it extends over pulleys 62 and 252, thus moving axis 242 towards axis 276. Axes 242 and 276 can, of course, define a plane in which the axes can move laterally toward and away from each other. As carriage 230 is moved leftward as belt 120b is wound onto spool 116b, belt 120a is unwound from about spool 116a of motor drive assembly 312a as the clutch in clutch housing 318a allows relative rotation between spool 116a and motor 314a, thereby lengthening the path over which belt 120a extends over pulleys 62 and 252.

The relationship between each pulley 62, 92, 252, or 254 and its associated retention roller 74, 104, or 256 is now discussed. There is a gap between the circumferential surface 330 of the pulley and the circumferential surface 332 of the retention roller through which belt 120 is fed. The retention roller prevents the belt 120 from becoming disengaged from the pulley during installation and operation of the lift or conveyor assembly 20, 20A, 220. Notably, the proximity of the circumferential surfaces 330, 332 of the pulley and retention roller is such that the gap therebetween is sufficient to allow a space between the outer surface 128 of belt 120 and the surface 332 of the retention roller. Preferably, the retention roller extends partially between the opposed flanges 70, 100 of the pulley, thereby capturing the belt between the flanges. Further, as readily understood from the above description, the reversing path of a belt 120 over a series of any three pulleys causes the belt to be directed at different return angles on the top and bottom sides of the pulleys. To accommodate this angling of the belt, the circumferential surface 330 of the pulley can be crowned.

As to a further discussion of the manner of usage and operation of the present invention, the same should be apparent from the above description. With respect to the above description then, it is to be realized that the optimum dimensional relationships for the parts of the invention, to include variations in size, materials, shape, form, function and manner of operation, assembly and use, are deemed readily apparent and obvious to one skilled in the art, and all equivalent relationships to those illustrated in the drawings and described in the specification are intended to be encompassed by the present invention.

## 13

Therefore, the foregoing is considered as illustrative only of the principles of the invention. Further, since numerous modifications and changes will readily occur to those skilled in the art, it is not desired to limit the invention to the exact construction and operation shown and described, and accordingly, all suitable modifications and equivalents may be resorted to, falling within the scope of the invention.

What is claimed is:

1. A scissors-type lift assembly comprising:
  - a base;
  - a platform, said lift assembly having elevated and lowered states in which said platform and said base are distant and proximate, respectively;
  - a pair of first and second scissor arms each having upper and lower ends respectively coupled to said platform and said base, said pair of scissor arms pivotably connected to each other intermediate their respective upper and lower ends about a central pivot axis;
  - a first pulley arrangement comprising at least one first pulley disposed about a first pulley axis;
  - a second pulley arrangement comprising at least one second pulley disposed about a second pulley axis, said first and second pulley axes disposed in a pulley plane and having lateral movement relative to each other as said lift assembly is moved between its said elevated and lowered states, said pulley plane fixed relative to said base;
  - a spool rotatable about an axis fixed to said base; and
  - a unitary belt guided through a path defined only by said first and second pulleys and said spool, said belt having opposed first and second ends, said belt first end engaged with said spool onto which said belt is wound and from which said belt is unwound;

said first and second pulley axes having a first lateral distance therebetween in said lift assembly elevated state and a second lateral distance therebetween greater than said first distance in said lift assembly lowered state, said first and second pulley axes pulled laterally together by said belt as it is wound onto said spool.
2. The scissors-type lift assembly of claim 1, wherein said pulley axes are biased laterally apart.
3. The scissors-type lift assembly of claim 1, wherein one of said first and second pulley axes is fixed relative to one of said first and second scissor arms.
4. The scissors-type lift assembly of claim 3, wherein the other of said first and second pulley axes is fixed relative to said base.
5. The scissors-type lift assembly of claim 4, wherein the other of said first and second scissor arms is pivotably attached to said base.
6. The scissors-type lift assembly of claim 1, further comprising a plurality of retention rollers, each of said retention rollers having a circumferential face disposed adjacent said belt with a positioning of said belt substantially along said path being maintained by said retention roller.
7. The scissors-type lift assembly of claim 6, wherein said plurality of retention rollers are disposed about a roller axis parallel with one of said first and second pulley axes.
8. A scissors-type lift assembly comprising:
  - a base;
  - a platform, said lift assembly having extended and contracted states, said platform at a relatively greater distance from said base in said extended state than in said contracted state;
  - a pair of first and second elongate arms each having opposite ends and extending between said base and said platform, said first and second arms pivotably connected together and crossing each other substantially at a first

## 14

- angle in said extended state and substantially at a second angle in said contracted state;
  - a first pulley arrangement comprising at least one first pulley rotatably disposed about a first pulley axis;
  - a second pulley arrangement comprising at least one second pulley rotatably disposed about a second pulley axis, said first and second pulley axes defining a stationary plane in which said first and second pulley axes have relative lateral movement, one of said first and second pulley arrangements affixed to one of, said first and second elongate arms, said arms moved between said first and second angles with relative lateral movement of said pulley axes;
  - a spool rotatable about an axis of rotation fixed relative to said base;
  - a unitary belt guided through a path defined only by said first and second pulleys and said spool, said belt having opposed first and second ends, said belt first end engaged with said spool onto which said belt is wound and from which said belt is unwound, said first and second pulley axes moved laterally in opposite directions relative to each other as said belt is respectively wound onto and unwound from said spool.
9. The scissors-type lift assembly of claim 8, further comprising a plurality of retention rollers, each of said retention rollers having a circumferential face disposed adjacent said belt with a positioning of said belt substantially along said path being maintained by said retention roller.
  10. The scissors-type lift assembly of claim 9, wherein said plurality retention disposed about a roller axis parallel with one of said first and second pulley axes.
  11. A lift assembly comprising:
    - a base;
    - a platform;
    - a pair of first and second elongate scissor arms each having a first end coupled to said base, and a second end coupled to said platform, said first and second scissor arms pivotably connected to each other at a location between their respective first and second ends; and
    - a belt and pulley apparatus comprising:
      - a unitary belt having opposite first and second ends,
      - a belt spool rotatably attached to said base and having reversible rotation about a spool axis, said belt first end secured to said spool, said belt second end secured to said base, said belt respectively wound onto and unwound from said spool with reversing rotation of said spool,
      - a first pulley arrangement comprising a plurality of first pulleys disposed about a first axis, and
      - a second pulley arrangement comprising at least one second pulley disposed about a second axis fixed relative to said base, said first and second axes substantially parallel with each other and laterally movable relative to each other in a stationary plane, said first axis fixed to one of said first and second scissor arms, with said unitary belt guided through a path defined only by said first and second pulley arrangements and said belt spool.
  12. The lift assembly of claim 11, wherein said first axis is located between the pivotal connection between said first and second arms and the first end of said one of said first and second scissor arms.
  13. The lift assembly of claim 11, further comprising a plurality of retention roller's, each of said retention rollers having a circumferential face disposed adjacent said belt with a positioning of said belt substantially along said path being maintained by said retention roller.

**15**

**14.** The lift assembly of claim **13**, wherein said plurality of retention rollers are disposed about a roller axis parallel with one of said first and second pulley axes.

**15.** A belt-driven transportation system comprising:

a plurality of first pulleys rotatably attached to a first member with each of said first pulleys defining a first belt engaging surface;

a pair of first flanges associated with each of said first pulleys disposed on opposite sides of said first belt engaging surface;

a plurality of second pulleys rotatably attached to a second member with each of said second pulleys defining a second belt engaging surface, said first and second members having relative movement to each other;

a pair of second flanges associated with each of said second pulleys disposed on opposite sides of said second belt engaging surface;

a unitary belt disposed about said first and second belt engaging surfaces and guided through a path defined by said first and second flanges, said path substantially lying in a stationary plane; and

a plurality of retention rollers with one of said retention rollers disposed adjacent each of said first and second

**16**

pulleys, said belt extending between said pulleys and said retention rollers with a positioning of said belt substantially on said path between said flanges being maintained by said plurality of retention rollers, said first and second members moving relative to each other with changes in a length of said path;

wherein one of said plurality of retention rollers are partially disposed between each of said pair of first and second flanges.

**16.** The belt-driven transportation system of claim **15**, wherein said plurality of first pulleys is rotatable about a first axis fixed relative to said first member, and said plurality of second pulleys is rotatable about a second axis fixed relative to said second member, said stationary plane defined by said first and second axes.

**17.** The belt-driven transportation system of claim **16**, wherein said plurality of retention rollers includes a first set of retention rollers rotatable about a third axis fixed relative to said first member and a second set of retention rollers rotatable about a fourth axis fixed relative to said second member.

\* \* \* \* \*

UNITED STATES PATENT AND TRADEMARK OFFICE  
**CERTIFICATE OF CORRECTION**

PATENT NO. : 8,662,477 B2  
APPLICATION NO. : 12/639632  
DATED : March 4, 2014  
INVENTOR(S) : Bacon

Page 1 of 1

It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

**IN THE CLAIMS:**

Column 14, line 31, please delete “plurality retention” before “disposed” and replace with -- plurality of retention rollers --

Column 14, line 64, please delete “roller’s” before “each” and replace with -- rollers --

Signed and Sealed this  
Tenth Day of June, 2014



Michelle K. Lee  
*Deputy Director of the United States Patent and Trademark Office*